Title: METHOD FOR MEASUREMENT OF ROTATION

RATES/ACCELERATIONS USING A ROTATION RATE CORIOLIS GYRO, AS WELL AS A CORIOLIS GYRO

WHICH IS SUITABLE FOR THIS PURPOSE IAP20 Rec'd PCT/PTO 23 JUN 2006

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BACKGROUND

Field of the Invention:

The <u>present</u> invention relates to <u>Coriolis</u>

<u>qyroscopes. More particularly, the invention pertains</u>

<u>to</u> a method for measurement of accelerations <u>with using</u>
a rotation rate Coriolis gyro, and to a Coriolis gyro

<u>which is</u> suitable for <u>such this</u> purpose.

15 <u>Description of the Prior Art</u>

Coriolis gyros (also referred to as

"vibration gyros") are being increasingly employed used for navigation. purposes; they have Such devices

include a mass system that which is caused to oscillate. The Each mass system generally has a large number of oscillation modes, which are initially independent of one another. In order A specific oscillation mode of the mass system is artificially excited to operate the Coriolis gyro. and this Such mode is referred to in the following text as the "excitation oscillation".

Coriolis forces occur that which draw energy from the excitation oscillation of the mass system When the Coriolis gyro is rotated and thus transmit a further oscillation mode of the mass system which is (referred to below in the following text as the "read

oscillation"). In order The read oscillation is tapped off to determine rotations of the Coriolis gyro and a corresponding read signal is investigated to determine whether any changes have occurred in the amplitude of the read oscillation which represent a measure of the rotation of the Coriolis gyro.

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Coriolis gyros may comprise either be in the form of both an open-loop or system and a closed-loop system. In a closed-loop system, the amplitude of the read oscillation is continuously reset to a fixed value (preferably zero) via respective control loops, and the resetting forces are measured.

The mass system of the Coriolis gyro which is also (referred to below in the following text as the "resonator") may in this case be of designed in widely differing designs. ways. For example, it is possible to use an integral mass system. Alternatively, it is possible to split the mass system into separate two oscillators which are coupled to one another via a spring system and capable of can carry out relative movements relative with respect to one another. High dimensional accuracies can be achieved particularly in particular with linear double-oscillator systems that which comprise a coupled system composed of two linear oscillators. In double-oscillator systems, the spring system that which couples the two linear oscillators to one another is, in general, designed so in such a way

that the two linear oscillators can be caused to oscillate along a first oscillation axis, with in which case the second oscillator additionally oscillating can additionally carry out oscillations along a second oscillation axis which is at right angles to the first oscillation axis. In such case, the movements of the second oscillator along the second oscillation axis can in this case be regarded as a read oscillation while those and the movements of the first and second oscillators along the first oscillation axis can be regarded as an excitation oscillation.

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Linear double-oscillator systems have the disadvantage that the oscillations of the two linear oscillators along the first oscillation axis can cause vibrations or reflections in the gyro frame. In this case, (The "gyro frame" should be understood to be a mechanical, non-oscillating structure in which the oscillators are "embedded", e.g., for example a non-oscillating part of a silicon wafer.) The vibrations or reflections in the gyro frame can, in turn, lead to disturbances (e.g. for example damping effects) to the oscillator movements. For example, the oscillations of the first and second linear oscillators along the first oscillation axis can thus be disturbed by both external vibrations and accelerations which act along the first oscillation axis. Analogously to this, external vibrations and accelerations acting which act in the direction of the second oscillation axis can

disturb the oscillations of the second linear oscillator along that this oscillation axis to corrupt the measured rotation rate which in precisely the same way as with all of the other disturbance influences mentioned. — leads to corruption of the measured rotation rate.

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SUMMARY AND OBJECTS OF THE INVENTION

It is therefore an The object of on which the present invention is based is to provide specify a Coriolis gyro by means of which any disturbance of the read oscillation (i.e., that is to say of the oscillation of the second linear oscillator in the direction of the second oscillation axis) as a result of the disturbance influences mentioned above can be largely avoided.

The present invention addresses the preceding
and other objects by providing, in a first aspect, In
order to achieve this object, the invention provides a
Coriolis gyro. as claimed in patent claim 1.

Furthermore, the invention provides a method for
measurement of accelerations/rotation rates using a
rotation rate Coriolis gyro as claimed in patent claim
7. Advantageous refinements and developments of the
idea of the invention can be found in the dependent
claims.
The Coriolis Such gyro according to the invention has a

first and a second resonator, which are each in the

form of a coupled system comprising a first and a second linear oscillator. with The first resonator is being mechanically/electrostatically connected/coupled to the second resonator such that the two resonators can be caused to oscillate in antiphase with respect to one another along a common oscillation axis.

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In a second aspect, For this reason, the invention provides a method for selective or simultaneous measurement of rotation rates and accelerations with This method uses a rotation rate Coriolis gyro that which has a first and a second resonator. The resonators which are each in the form of a coupled system comprising a first and a second linear oscillator. and in which Rotation rates to be determined are determined by tapping and evaluation of the deflections of the second oscillators. The method has the following steps:

In such method, the two resonators are caused to carry out oscillations in antiphase with one another along a common oscillation axis. The deflections of the second oscillators are compared with one another in order to determine an antiphase deflection component that which is a measure of the rotation rate to be measured and/or in order to determine a common in-phase deflection component, which is a measure of the acceleration to be measured. and calculation of The rotation rate/acceleration to be measured is then

calculated from the in-phase deflection
component/anti-phase deflection component.

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The preceding and other features of the invention will become further apparent from the detailed description that follows. Such description is accompanied by a set of drawing figures. Numerals of the drawing figures, corresponding to those of the written description, point to the features of the invention with like numerals referring to like features of the invention throughout.

BRIEF DESCRIPTION OF THE DRAWINGS

Figure 1 is a schematic illustration of shows one possible embodiment of a mass system having which comprises two linear oscillators, with corresponding control loops, for exciting which are used to excite the first oscillator.

Figure 2 <u>is a scematic illustration of a</u> shows one possible embodiment of a mass system <u>having</u> which comprises two linear oscillators with corresponding measurement and control loops for a rotation rate Ω and a quadrature bias B_Q , as well as auxiliary control loops for compensation of the quadrature bias B_Q .

Figure 3 is a schematic illustration shows an

embodiment of according to the invention, which comprises four linear oscillators, with corresponding measurement and control loops for a rotation rate Ω and a quadrature bias B_0 , as well as the auxiliary control loops for compensation of the quadrature bias.

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Figure 4 <u>is a block diagram of an embodiment</u>
of a shows one preferred embodiment of the control
system for incorporation into a mass system in
accordance with that illustrated in Figure 3 above.
shown in Figure 3.

DETAILED DESCRIPTION OF THE PREFERRED EMBODIMENT

Accordingly, the A Coriolis gyro according to the invention has a mass system that which comprises two double-oscillator systems that is to say (two resonators) or four linear oscillators. The Antiphase oscillations of the two resonators with respect to one another in this case result in the center of gravity of the mass system remaining stationary if the two resonators are designed appropriately. This results in the oscillation of the mass system producing no not being able to produce any external vibrations that which in turn would result in disturbances in the form of damping/reflections. Furthermore, External vibrations and accelerations in the direction of the common oscillation axis have no influence on the antiphase movement of the two resonators along the

common oscillation axis.

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The first resonator can be coupled to the second resonator (e.g., for example via a spring system that which connects the first resonator to the second resonator). A further possibility is to couple the first resonator to the second resonator via an electrostatic field. Both types of coupling may be used either preclusively on their own or in combination. conjunction. It is sufficient, for example, for both resonators to be formed on in a common substrate so that the mechanical coupling is replaced by a mechanical connection which is itself provided by the common substrate.

second resonators resonator are preferably identical in terms of mass and shape. In this case, The two resonators may be arranged axially symmetrically with respect to one another with reference respect to an axis of symmetry which is at right angles to the common oscillation axis. That is, to say the first resonator is mapped by the axis of symmetry onto the second resonator. However, The invention is not restricted to this and it is sufficient for the two resonators to have the same mass, but to be designed with different shapes.

As already mentioned, the coupled resonators

are designed so in such a way that both linear oscillators of a resonator can be caused to oscillate in antiphase along a first oscillation axis (excitation oscillation). and The second linear oscillator can additionally be caused to oscillate along a second oscillation axis (read oscillation). If the first and the second oscillation axes are at right angles to one another, and both resonators are caused to oscillate in antiphase with respect to one another along the first oscillation axis (common oscillation axis), then the second oscillators are deflected in the opposite direction during rotation of the Coriolis gyro (antiphase deflection). while, In contrast, during acceleration of the Coriolis gyro, the second linear oscillators are deflected in the same direction (in-phase deflection). It is thus possible to measure accelerations or rotations selectively. The acceleration is measured by evaluation of an in-phase oscillation, and the rotation rate is measured by evaluation of an antiphase oscillation. In the following text, The expressions "in-phase" and "antiphase" have the following meanings: if the coordinates in the excitation direction are denoted \mathbf{x} and those in the read direction are denoted y, then $x_1 = x_2$, $y_1 = y_2$ for in-phase oscillation and $x_1 = -x_2$, $y_1 = -y_2$ for antiphase oscillation (in this case, the index "1" denotes the first oscillator, and the index "2" the second oscillator).

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The common in-phase deflection component is determined advantageously as follows: a first quadrature bias which occurs within the first resonator and a second quadrature bias which occurs within the second resonator are determined. The first and the second quadrature biases are then added and subtracted in order to determine a common quadrature bias component (in-phase component) and a difference quadrature bias component (antiphase component). The common quadrature bias component is proportional to the acceleration to be measured and corresponds to the common in-phase deflection component. The difference quadrature bias component (difference) corresponds to the antiphase deflection component. The Rotation rate can thus be measured at the same time as the acceleration, via the difference quadrature bias component.

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In order to assist understanding of the acceleration measurement principle described above, the physical principles of a Coriolis gyro will be briefly explained once again below in the following description, using the example of a linear double-oscillator system. In general, Coriolis gyros have a quadrature bias (i.e., that is to say a zero error). The quadrature bias is in this case composed of a plurality of quadrature bias components. One of these quadrature bias components arises from alignment errors of the first and second linear oscillator with respect

to one another, with <u>such</u> these alignment errors being unavoidable <u>due</u> to because of manufacturing tolerances. The alignment errors between the two oscillators produce a zero error in the measured rotation rate signal.

The Coriolis force can be represented as:

 $\vec{F} = 2m\vec{v}, x\hat{\Omega}$

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 $ec{F}$ Coriolis force

[1]

- m Mass of the oscillator
- $ec{v}$, Velocity of the oscillator
- $ar{\Omega}$ Rotation rate

If the mass <u>that which</u> reacts to the Coriolis force is equal to the oscillating mass, and <u>if</u> the oscillator is operated at the natural frequency ω , then:

$$2m\vec{v}_s x\vec{\Omega} = m\vec{a}_c \tag{2}$$

The oscillator velocity is given by:

$$\vec{V}_s = \vec{V}_{s0} \sin \omega t \tag{3}$$

where

 \overline{v}_{s0} oscillator amplitude

 ω = natural frequency of the oscillator

The oscillator and Coriolis accelerations are

thus given by:

$$\vec{a}_s = \vec{v}_{s0} \omega \cos \omega t$$

 $\vec{a}_c = 2 \vec{v}_{s0} \sin \omega t \times \vec{\Omega}$

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The two acceleration vectors are thus spatially at right angles to one another and are offset through 90° with respect to one another in the time function (spatial and time orthogonality).

These two criteria can be employed used in order to separate the oscillator acceleration \vec{a}_s from the Coriolis acceleration \vec{a}_c . The ratio of the abovementioned acceleration amplitudes a_c and a_s is:

$$\frac{a_c}{a_i} = \frac{2\Omega}{\omega} \tag{5}$$

If the rotation rate is $\Omega=5^{\circ}/h$ and the natural frequency of the oscillator is $f_{s}=10$ KHz, then:

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$$\frac{a_c}{a_s} = 7.7 \cdot 10^{-10}$$
 [6]

For an accuracy of 5°/h, undesirable couplings of the first oscillator to the second oscillator must not exceed 7.7 • 10⁻¹⁰, or must be constant. at this value If a mass system composed of two linear oscillators is used, which are coupled to one another via spring elements is employed, then the

accuracy of the spatial orthogonality between the oscillation mode and the measurement mode is limited due to because of the alignment error of the spring elements. The Achievable accuracy (limited by manufacturing tolerances) is 10^{-3} to 10^{-4} . The accuracy of the Time orthogonality accuracy is limited by the phase accuracy of the electronics at, for example, 10 KHz, which can likewise be complied with only to at most 10^{-3} to 10^{-4} . This means that the ratio of the accelerations as defined above cannot be satisfied.

Realistically, the resultant error in the measured acceleration ratio a_c/a_s is:

$$\frac{a_c}{a_c} = 10^{-6} \text{ to } 10^{-8}$$

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The spatial error results in a so-called quadrature bias B_Q , which, together with the time phase error Δ_{φ} , results in a bias B:

$$B_Q=6.5 \cdot 10^6$$
 °/h to $6.5 \cdot 10^5$ °/h
$$\Delta_{\phi}=10^{-3} \text{ to } 10^{-4}$$

$$B=B_Q \cdot \Delta_{\phi}=6,500$$
 °/h to 65 °/h [8]

20 limitation restriction to the measurement accuracy. In this case, it should be noted that the preceding above error analysis takes account only of the direct coupling of the oscillation mode to the read mode. Further quadrature bias components also exist and occur, for example, as a result of couplings with other

oscillation modes.

If the Coriolis gyro is designed so in such a way that the first oscillators are connected by first spring elements to a gyro frame of the Coriolis gyro, and the second oscillators are connected by second spring elements to in each case one of the first oscillators, then the acceleration to be measured results in a change in the mutual alignment of the first oscillators with respect to the second oscillators. and This is particularly in particular manifested in a change in the alignment of the second spring elements. The alignment change of the second spring elements in this such case produces an "artificial" quadrature bias component (i.e., that is to say an "error") in the quadrature bias signal. It is thus also indirectly possible to use the determination of the quadrature bias to deduce the acceleration to be measured, which produces the corresponding "artificial" quadrature bias component.

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The alignments of the first and second spring elements are preferably at right angles to one another. The spring elements may have any desired shape. The expression "first quadrature bias" and "second quadrature bias" in each case preferably mean the total quadrature bias of a resonator. However, It is also possible in the acceleration measurement method according to the invention to in each case determine

only one quadrature bias component in each resonator.

In <u>such which</u> case the determined quadrature bias component must include at least <u>the that</u> component which is produced by the acceleration to be measured or the rotation to be measured.

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The Coriolis gyro preferably has a device for determination of first rotation rate and quadrature bias signals that which occur within the first resonator, and second rotation rate and quadrature bias signals that which occur within the second resonator. Furthermore, the Coriolis gyro may have a device for production of electrostatic fields, by means of which the alignment angle of the first spring elements with respect to the gyro frame can be varied and/or the alignment angle of the second spring elements can be varied with respect to the first oscillators. The alignment/strength of the electrostatic fields can then be regulated by provision of appropriate control loops so such that the first and the second quadrature bias are in each case as small as possible. A computation unit can use the first and second rotation rate/quadrature bias signals to determine the rotation rate, and can use an in-phase component of the electrostatic fields which compensate for the first and second quadrature biases, to deduce the acceleration to be measured.

The quadrature bias is thus preferably

eliminated at its point of origin. itself, that is to say mechanical alignment errors of the two oscillators with respect to one another and changes in the mutual alignment of the two oscillators caused by the acceleration/rotation to be measured are compensated for by means of an electrostatic force produced by the electrostatic field that which acts on one or both oscillators. and is This type of quadrature bias compensation has the advantage that both rotation rates and accelerations can be determined with increased measurement accuracy.

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In one particularly preferred embodiment, the electrical fields change the alignment angles of the first and second spring elements in order to make the alignments of the first and second spring elements orthogonal with respect to one another. Orthogonalization such as this results in compensation for the quadrature bias (component) produced. in this way Further contributions to the quadrature bias are used to set the error angle with respect to orthogonality so such that the overall quadrature bias disappears. The alignment angles of the second spring elements with respect to the first oscillator are preferably varied by means of the electrostatic field, and the alignment angles of the first spring elements with respect to the gyro frame of the Coriolis gyro are not changed. However, It is also possible to use the electrostatic field to vary only the alignment angles

of the first spring elements or to vary the alignment angles of both the first and the second spring elements.

One particularly preferred embodiment of a Coriolis gyro according to the invention has:

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- an ("overall") resonator, which is in the form of a system comprising two coupled first (linear) oscillators ("sub-resonators") which are excited in antiphase and each containing contain a second linear read oscillator,
- a device for production of at least one electrostatic field, by means of which the alignment of the two coupled first oscillators with respect to the second (read) oscillators can be varied,
- a device for determination of the quadrature biases of the read oscillators that which are caused by alignment errors of the two oscillators with respect to the excitation oscillator and further coupling mechanisms,
- a control loop which in each case regulates the intensity of the at least one electrostatic field by means of at least one corresponding control signal such that the determined quadrature biases are as small as possible,
- a computation unit, which in each case forms differences and sums of the at least one control signal and uses them to determine the rotation rate and the acceleration.

In principle, it is possible to calculate accelerations and rotation rates solely just on the basis of the determined quadrature biases. that is to say It is not absolutely essential to compensate for the first and second quadrature bias in order to determine the quadrature biases. However, this is advisable for measurement accuracy reasons, since as phase tolerances results in mixing the rotation rate and the quadrature being mixed with one another. The invention covers both alternatives.

It has also been found to be advantageous for each of the second oscillators to be attached to or clamped in on the first oscillator "at one end" in the resonators. "Clamped in at one end" can in this case be understood not only in the sense of the literal wording but also in a general sense. In general, attached or clamped in at one end means that the force is introduced from the first oscillator to the second oscillator essentially from one "side" of the first oscillator. If, by way of for example, the oscillator system were to be designed in such a way that the second oscillator were is bordered by the first oscillator and is connected to it by means of second spring elements, then the expression clamped in or attached at one end would imply the following: the second oscillator is readjusted for the movement by the first oscillator, by the first oscillator alternately

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"pushing" or "pulling" the second oscillator by means of the second spring elements.

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Clamping the second oscillator in at one end on the first oscillator has the advantage that, when an electrostatic force is exerted on the second oscillator as a result of the alignment/position change of the second oscillator which results from this, the second spring elements can be slightly curved, thus making it possible, without any problems, to vary the corresponding alignment angle of the second spring elements. If the second oscillator in this example were to be attached to additional second spring elements so in such a way that, during movement of the first oscillator, the second oscillator were at the same time to be "pulled" and "pushed" by the second spring elements, then this would be equivalent to the second oscillator being clamped in or attached "at two ends" to the first oscillator (with the force being introduced to the second oscillator from two opposite ends of the first oscillator). In such this case, the additional second spring elements would produce corresponding opposing forces when an electrostatic field is applied, so that changes in the alignment angles of the second spring elements could be achieved only with difficulty. However, clamping in at two ends is acceptable when the additional second spring elements are designed so such that the influence of these spring elements is small so that all of the

spring elements can bend without any problems. in this case as well, That is, to say the clamping in is effectively at one end.

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Depending on the design of the oscillator, structure clamping in at one end can be effectively provided just by the "influence" (force introduction) of the additional second spring elements being 40% or less. However, this value does not present any limitation on restriction to the invention. and It is also feasible for the influence of the second spring elements to be more than 40%. By way of For example, clamping in at one end can be achieved by all of the second spring elements that which connect the second oscillator to the first oscillator being arranged parallel and on the same plane. as one another All start and end points of the second spring elements are in each case attached to the same ends of the first and second oscillator. The start and end points of the second spring elements may in this case each advantageously be on a common axis, with the axes intersecting the second spring elements at right angles.

If the second oscillator is attached to or clamped on the first oscillator at one end, then the first spring elements are preferably designed to such that they clamp the first oscillator in on the gyro frame at two ends (the expressions "at one end" and "at

two ends" can be used analogously here). As an alternative, to this however, it is possible for the spring elements also to be designed to in such a way that they clamp in the first oscillator at one end. By 5 way of For example, all the first spring elements that which connect the first oscillator to the gyro frame of the Coriolis gyro can be arranged parallel and on the same plane as one another, with the start and end points of the first spring elements in each case 10 preferably being located on a common axis. It is equally possible for the spring elements to be designed so in such a way that the first oscillator is clamped in on the gyro frame at one end, and the second oscillator is clamped in at two ends by the first oscillator. It is also possible for both oscillators to 15 be clamped in at two ends. For quadrature bias compensation, it has been found to be advantageous for at least one of the two oscillators to be clamped in at one end.

The invention will be explained in more detail in the following text with reference to one exemplary embodiment in the figures, in which:

Figure 1 <u>illustrates</u> shows the schematic

25 design of a linear double oscillator 1 with

corresponding electrodes <u>including</u> as well as a block

diagram of associated evaluation/excitation electronics

2. The linear double oscillator 1 is preferably

produced by means of etching processes from a silicon wafer. and It has a first linear oscillator 3, a second linear oscillator 4, first spring elements 5_1 to 5_4 , second spring elements 6_1 and 6_2 as well as parts of an intermediate frame 7_1 and 7_2 and of a gyro frame 7_3 and 7_4 . The second oscillator 4 is mounted within the first oscillator 3 to such that it can oscillate, and is connected to it via the second spring elements 6_1 , 6_2 . The first oscillator 3 is connected to the gyro frame 7_3 , 7_4 by means of the first spring elements 5_1 to 5_4 and the intermediate frame 7_1 , 7_2 .

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Furthermore, First excitation electrodes 8₁ to 8₄, first read electrodes 9₁ to 9₄, second excitation electrodes 10₁ to 10₄, and second read electrodes 11₁ and 11₂ are also provided. All of the electrodes are mechanically connected to the gyro frame, although but are electrically isolated. (The expression "gyro frame" refers to means a mechanical, non-oscillating structure in which the oscillators are "embedded", e.g., for example the non-oscillating part of the silicon wafer).

When If the first oscillator 3 is excited by means of the first excitation electrodes 8_1 to 8_4 to oscillate in the X1 direction, such then this movement is transmitted through the second spring elements 6_1 , 6_2 to the second oscillator 4 (alternate "pulling" and "pushing"). The vertical alignment of the first spring elements 5_1 to 5_4 prevents the first oscillator 3 from

moving in the X2 direction. However, $\frac{1}{2}$ vertical oscillation can be carried out by the second oscillator 4 as a result of the horizontal alignment of the second spring elements 6_1 , 6_2 . When corresponding Coriolis forces accordingly occur, then the second oscillator 4 is excited to oscillate in the X2 direction.

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A read signal that which is read from the first read electrodes 9_1 to 9_4 and $\frac{1}{100}$ proportional to the amplitude/frequency of the X1 movement of the first oscillator 3 is supplied, via appropriate amplifier elements 21, 22 and 23, to an analog/digital converter 24. An appropriately digitized output signal from the analog/digital converter 24 is demodulated not only by a first demodulator 25 and but also by a second demodulator 26 to form corresponding output signals, with the two demodulators operating with an offset of 90° with respect to one another. The output signal from the first demodulator 25 whose output signal controls a frequency generator 30 so such that the signal occurring which occurs downstream from the demodulator 25 is regulated at zero is supplied to a first regulator 27 in order to regulate the frequency of the excitation oscillation (the oscillation of the mass system 1 in the X1 direction). Analogously to this, the output signal from the second demodulator 26 is regulated at a constant value which is (predetermined by the electronics component 291. A second regulator 31 insures ensures that the amplitude of the excitation

oscillation is regulated. The output signals from the frequency generator 30 and from the amplitude regulator 31 are multiplied by one another at by means of a multiplier 32. An output signal from the multiplier 32, which is proportional to the force to be applied to the first excitation electrodes 8, to 8, acts not only on a first force/voltage converter 33 but also on a second force/voltage converter 34, which use the digital force signal to produce digital voltage signals. The digital output signals from the force/voltage converters 33, 34 are converted by via a first and a second digital/analog converters converter 35, 36 to corresponding analog voltage signals. which are Such signals are then passed to the first excitation electrodes 8_1 to 8_4 : The first regulator 27 and the second <u>regulators</u> regulator 27, 31 readjust the natural frequency of the first oscillator 3 and set the amplitude of the excitation oscillation to a specific, predeterminable value.

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When Coriolis forces occur, <u>resultant</u> the movement of the second oscillator 4 in the X2 direction (read oscillation) that results from this is detected by the second read electrodes 11_1 , 11_2 , and a read signal, which is proportional to the movement of the read oscillation, is supplied via appropriate amplifier elements 40, 41 and 42 to an analog/digital converter 43 (see Figure 2). A digital output signal from the analog/digital converter 43 is demodulated by a third

demodulator 44 in phase with the direct-bias signal and is demodulated by a fourth demodulator 45, offset through 90°. A corresponding output signal from the first demodulator 44 is applied to a third regulator 46, whose output signal is a compensation signal that and corresponds to the rotation rate Ω to be measured. An output signal from the fourth demodulator 45 is applied to a fourth regulator 47 whose output signal is a compensation signal and is proportional to the quadrature bias to be compensated. for The output signal from the third regulator is modulated by means of a first modulator 48, and the output signal from the fourth regulator 47 is modulated in an analogous manner to this by means of a second modulator 49, so that amplitude-regulated signals are produced whose frequencies correspond to the natural frequency of the oscillation in the X1 direction ($\sin \approx 0^{\circ}$, $\cos \approx 90^{\circ}$). Corresponding output signals from the modulators 48, 49 are added in an addition stage 50, whose output signal is supplied both to a third force/voltage converter 51 and to a fourth force/voltage converter 52. The corresponding output signals for the force/voltage converters 51, 52 are supplied to digital/analog converters 53, 54, whose analog output signals are applied to the second excitation electrodes 10_2 to 10_3 , and reset the oscillation amplitudes of the second oscillator 4.

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The electrostatic field which is produced by

the second excitation electrodes 10_1 and 10_4 (or the two electrostatic fields which are produced by the electrode pairs 10_1 , 10_3 and 10_2 , 10_4) results in an alignment/position change of the second oscillator 4 in the X2 direction, and thus in a change in the alignments of the second spring elements 6_1 to 6_2 . The fourth regulator 47 regulates the signal which is applied to the second excitation electrodes 10_1 and 10_4 so in such a way that the quadrature bias which is included in the compensation signal of the fourth regulator 47 is as small as possible, or disappears. A fifth regulator 55, a fifth and a sixth force/voltage converter 56, 57 and two analog/digital converters 58, 59 are used for this purpose.

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supplied to the second excitation electrode 10_1 or (alternatively to electrode 11_1). The analog output signal from the analog/digital converter 59 is supplied to the second excitation electrode 10_4 or (alternatively to electrode 11_2).

As Since the second oscillator 4 is clamped $\frac{1}{10}$ only by the second spring elements 6_1 to 6_2 (clamping <u>clamped</u> in at one end), <u>such</u> the alignment of the these spring elements can be varied without problem by the electrostatic field any problems. It is additionally also possible to provide additional second spring elements, which result resulting in the second oscillator 4 being clamped in at two ends, provided that <u>such</u> these additional spring elements are appropriately designed to insure ensure that clamping in at one end is effective. effectively achieved. In order to permit allow the same effect for the spring elements 5_1 , 5_2 (and for the spring elements 5_3 , 5_4 as well) the third and fourth spring elements 5_3 , 5_4 , as well as and the first and second spring elements 5_1 , 5_2 may be omitted, thus resulting in the first oscillator 3 being clamped in at one end (together with an appropriately modified electrode configuration, which is not shown here). In such a situation such as this, the second oscillator 4 could may also be attached to the first oscillator by means of further spring elements in order to achieve clamping in at two ends.

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A One preferred embodiment of the Coriolis gyro of according to the invention as well as and its method of operation will be described in more detail $\frac{1}{10}$ the following description with reference to Figure 3, a schematic illustration of a mass system comprising four linear oscillators with corresponding measurement and control loops for rotation rate and quadrature mass, as well as auxiliary control loops for compensation of the quadrature bias. Figure 3 shows The schematic layout of coupled system 1' comprises comprising a first resonator 70_1 and a second resonator 70_2 . The first resonator 70, is coupled to the second resonator 70, by via a mechanical coupling element (a spring) 71. The first and the second resonator 70_1 , 70_2 are formed in a common substrate and may can be caused to oscillate in antiphase with respect to one another along a common oscillation axis 72. The first and the second resonators resonator 70_1 , 70_2 are identical, and are mapped onto one another via an axis of symmetry 73. The design of the first and $\frac{1}{100}$ second resonator 70_1 , 70_2 has already been explained in conjunction with Figures 1 and 2 and will therefore not be explained again. (Identical and mutually corresponding components or component groups are identified by the same reference numbers with identical components which are associated with different resonators being identified by different indices.)

 $\underline{\mathbf{A}}$ One major difference between the double

oscillators shown in Figure 3 and those the double oscillators shown in Figures 1 and 2 is that some of the individual electrodes are physically combined to form one overall electrode. For example, the individual electrodes which are identified by the reference numbers 8₁, 8₂, 9₁ and 9₂ in Figure 3 thus form a common electrode. Further, Furthermore, the individual electrodes which are identified by the reference numbers 8₃, 8₄, 9₃ and 9₄ form a common electrode, and those with the reference numbers 10₄, 10₂, 11₂ as well as the reference numbers 11₁, 10₃ and 10₁ each form an overall electrode. The same applies in an analogous manner to the other double-oscillator system.

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During operation of the coupled system 1' in

accordance with according to the invention, the two resonators 70₁, 70₂ oscillate in antiphase along the common oscillation axis 72. The coupled system 1' is thus not susceptible to external disturbances or to those disturbances which are emitted by the coupled system 1' itself into the substrate in which the resonators 70₁ and 70₂ are mounted.

When the coupled system 1' is rotated, then the second oscillators 4_1 and 4_2 are deflected in mutually opposite directions (i.e., in the X2 direction and in the opposite direction to the X2 direction). When an acceleration of the coupled system 1' occurs, then the second oscillators 4_1 , 4_2 are each deflected in

the same direction, <u>i.e.</u> $\frac{1}{2}$ specifically in the same direction as the acceleration provided that such this acceleration is in the X2 direction, or in the opposite direction. to it Accelerations and rotations can thus be measured simultaneously or selectively. Quadrature bias compensation can be carried out at the same time during the measurement process in the resonators 70_1 , 70_2 . However, this is not absolutely essential.

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In principle, it is possible to operate the 10 coupled system 1' on the basis of the evaluation/excitation electronics 2 described with reference to in Figures 1 and 2. However, An alternative method (carrier frequency method) is used instead <u>used</u> of this in the embodiment of shown in 15 Figure 3. <u>Such This</u> operating method will be described below. in the following text.

The evaluation/excitation electronics 2 which are identified by the reference number 2' include have three control loops: a first control loop for 20 excitation and/or control of an antiphase oscillation of the first oscillators $\mathbf{3}_1$ and $\mathbf{3}_2$ along the common oscillation axis 72, a second control loop for resetting and compensation of the oscillations of the second oscillator $\mathbf{4}_1$ along the X2 direction, and a control loop for resetting and compensation of the oscillations of the second oscillator 4_2 along the X2direction. The three described control loops include

have an amplifier 60, an analog/digital converter 61, a signal separation module 62, a first to third demodulation module 63_1 to 63_3 , a control module 64, an electrode voltage calculation module 65, a carrier frequency addition module 67, and a first to sixth digital/analog converter 66_1 to 66_6 .

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Carrier frequencies can be applied to the electrodes 8_1 to 8_8 , 9_1 to 9_8 , 10_1 to 10_8 and 11_1 to 11_4 for tapping excitation of the antiphase oscillation or of the oscillations of the second oscillators 4_1 , 4_2 . This may be accomplished in a number of ways. They include: a) using three different frequencies, with one frequency being associated with each control loop, b) using square-wave signals with a time-division multiplexing method, and or c) using random phase scrambling (stochastic modulation method).

The carrier frequencies are applied to the electrodes 8_1 to 8_8 , 9_1 to 9_8 , 10_1 to 10_8 and 11_1 to 11_4 via the associated signals UyAo, UyAu (for the second oscillator 4_1), and Ux1, Uxr (for the antiphase resonance of the first oscillators 3_1 to 3_2) and as well as UyBu and UyBo (for the second oscillator 4_2) that which are produced in the carrier frequency addition module 67 and are excited in antiphase with respect to the abovementioned frequency signals. The oscillations of the first and second oscillators 3_1 , 3_2 , 4_1 and 4_2 are tapped off via those parts of the gyro frame which are

identified by the reference numbers 7_7 , 7_9 , 7_{11} and 7_{13} , and in this case are additionally (used as tapping electrodes in addition to their function as suspension points for the mass system). For this purpose, the two resonators 70_1 , 70_2 are preferably and advantageously designed to be electrically conductive, with all of the frames, springs and connections. The signal, which is tapped off by means of the gyro frame parts 7_7 , 7_9 , 7_{11} and 7_{13} and is supplied to the amplifier 60, contains information about all three oscillation modes. It and is converted by the analog/digital converter 61 to a digital signal which is supplied to the signal separation module 62.

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The assembled signal is separated into three 15 different signals in the signal separation module 62: x(which contains information about the antiphase oscillation), yA (which contains information about the deflection of the second oscillator 4_1) and as well as yB (which contains information about the deflection of 20 the second oscillator $\mathbf{4}_{2}$). The signals are separated differently in accordance with depending on the type of carrier frequency method used (see a) to c) above). and Separation is carried out by demodulation with the corresponding signals of the carrier frequency method 25 that is used. The signals x, yA and yB are supplied to the demodulation modules 63_1 to 63_3 that which demodulate them with using an operating frequency of the antiphase oscillation for 0° and 90°. The control

module 64 and as well as the electrode voltage calculation module 65 for regulation/calculation of the signals Fxl/r or Uxl/r, respectively, are preferably configured analogously to the electronics module 2 of shown in Figure 1. The control module 64 and the electrode voltage calculation module 65 (for regulation/calculation of the signals FyAo/u, UyAo/u, and FyBo/u, UyBo/u) are preferably designed analogously to the electronics module 2 of shown in Figure 2.

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10 Figure 4 is a block diagram of an embodiment of a control system for incorporation into a mass system in accordance with Figure 3. It shows one preferred embodiment of the control system that is identified by the reference number 64 in Figure 3. The 15 control system 64 $\underline{includes}$ \underline{has} a first to third part 64_1 to 64_3 . The first part 64_1 has a first regulator 80, a frequency generator 81, a second regulator 82, an electronics component 83, an addition stage 84 and a multiplier 85. The method of operation of the first 20 part corresponds essentially to that the method of operation of the electronics module 2 of shown in Figure 1 and will therefore not be described once again here. The second part 64_2 has a first regulator 90, a first modulator 91, a second regulator 92, a second 25 modulator 93 and a third regulator 94. A first and a second addition stage 95, 96 are also provided. A rotation rate signal $\boldsymbol{\Omega}$ can be determined at the output of the first regulator 90, and an assembled signal

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comprising a quadrature bias $B_{\rm Q}$ and an acceleration A can be determined at the output of the third regulator 94.

The third part 64_3 of the control system 64has a first regulator 100, a first modulator 101, a 5 second regulator 102, a second modulator 103 and a third regulator 104. A first and a second addition stage 105, 106 are also provided. A rotation rate signal Ω with $\frac{1}{2}$ negative mathematical sign can be 10 tapped off at the output of the first regulator 100 and an assembled signal comprising the quadrature bias $\boldsymbol{B}_{\boldsymbol{Q}}$ with $\frac{1}{2}$ negative mathematical sign and an acceleration signal A can be tapped off at the output of the third regulator 104. The method of operation of the second 15 and of the third parts part 64_2 and 64_3 corresponds to that of the electronics module 2 illustrated in Figure 2, and will therefore not be explained once again here.

Only the signals for resetting of the rotation rate and the quadrature, after the

multiplication by the operating frequency, are passed, together with the DC voltages for the quadrature auxiliary regulator, to a combined electrode pair. The two signals are therefore added so that the calculation of the electrode voltages includes the reset signals for the oscillation frequency and the DC signal for quadrature regulation. The electrode voltages Uxl/r, UyAo/u and UyBo/u thusly calculated in this way are

then added to the carrier frequency signals and $\frac{1}{2}$ jointly passed via the analog/digital converters 66_1 to 66_6 to the electrodes.

The carrier frequency methods described above with antiphase excitation have the advantage that a signal is applied to the amplifier 60 only when the linear oscillators 3_1 , 3_2 , as well as 4_1 and 4_2 , are deflected. The frequency signals which are used for excitation may be discrete frequencies or square-wave signals. Square-wave excitation is preferred, as it is easier to produce and process.

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A number of analyses relating to the measurement accuracy of the acceleration measurement method according to the invention are will also be described in the following text.

The Rotation rate results in an antiphase deflection of the oscillators 4_1 and 4_2 at the operating frequency of the Coriolis gyro. In contrast, acceleration results in an in-phase deflection of the oscillators 4_1 and 4_2 with in which case the acceleration can be measured in the frequency range from 0 Hz to about 500 Hz with a measurement accuracy of 50 mg to 50 lg.

The in-phase deflection to be measured is given by:

 $\alpha = \frac{a}{\ell \cdot \omega^2}$

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- α Deflection angle
- a Acceleration
- ! Length of the spring
- ω Natural frequency of the oscillators 4_1 to 4_2 .

For typical natural frequencies $\omega=2$ * $\pi f=6000$ rad/s to 60000 rad/s and spring lengths of $\mathcal{L}=1$ mm of Coriolis gyros, the measurement accuracy of, for example 5 mg is:

 $\alpha = 1.4*10^{-6}$ to $1.4*10^{-8}$ rad or $x_2=x_1=1.4$ nm to 14 pm.

Small deflections such as the above these are difficult to measure in the frequency range from 0 to 500 Hz. At a minimum the least, this requires additional electronic complexity is required for the multisensor according to the invention as because the electronics have to measure very accurately in both the operating range of the gyro function (rotation rate measurement) from 1 to 10 KHz and in the operating range for measurement of the acceleration from 0 to 500 Hz.

This disadvantage can be overcome <u>in</u>

according to the invention by using the quadrature

regulation, as described above, for a mass system

comprising two linear oscillators (Figures 1 and 2) for
the mass system composed of four linear oscillators

(Figure 3): the acceleration detunes the orthogonality error, thus resulting in an in-phase quadrature signal, which can clearly be seen, at the operating frequency in the oscillators 4_1 and 4_2 :

$$Q_Q = \frac{a_Q}{a_s} \cdot \frac{\omega}{2} = \alpha \frac{\omega}{2}$$

In this case, Ω_Q is the quadrature rotation rate, a_Q is the quadrature acceleration and a_s is the oscillator acceleration.

For a measurement accuracy of, for example 5 mg $(\alpha = 1.4.10^{-6} \text{ rad})$, this results in:

$$\Omega_{\rm Q}=0.0042\frac{rad}{s}=0.25\,^{\circ}/{\rm s=866\,^{\circ}/h}$$
 at a natural frequency of 1 kHz $\Omega_{\rm Q}=4.2\cdot 10^{-5}\frac{rad}{s}/0.0025\,^{\circ}/{\rm s=8.7\,^{\circ}/h}$ at a natural frequency of 10 kHz

For a rotation rate sensor of 5°/h, the quadrature rotation rate of 866°/h can be verified with certainty using the same electronics. while, In contrast, at the natural frequency of 10 KHz and with the quadrature rotation rate of 8.7°/h, the verification limit of the rotation rate sensor of 5°/h is virtually exhausted. Although this measurement is also stable in the long run term, it depends on the long-term stability of the quadrature rotation rate. The actual quadrature rotation rate is an antiphase

signal. The stability of the acceleration measurement therefore depends on the difference in the quadrature rotation rates from the oscillator 4_1 to the oscillator 4_2 , and their stability. Since the two oscillators are located close to one another and were manufactured in one process step, it is predicted that it is possible to cover a range with low accuracy from 50 mg to 50 μg .

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While this invention has been described with reference to its presently preferred embodiment, it is not limited thereto. Rather, the invention is limited only insofar as it is defined by the following set of patent claims and includes within its scope all equivalents thereof.

Patent Claims

What is claimed is:

- A Coriolis gyro (1'), having a first and a second
- 2 resonator $(70_1, 70_2)$, which are each in the form of a
- 3 coupled system comprising a first and a second linear
- 4 oscillator $(3_1, 3_2, 4_1, 4_2)$, with the first resonator
- 5 (70_1) being mechanically/electrostatically
- 6 connected/coupled to the second resonator (70_2) such
- 7 that the two resonators can be caused to oscillate in
- 8 antiphase with one another along a common oscillation
- 9 axis (72).
- 1 2. The Coriolis gyro (1') as claimed in claim 1,
- 2 characterized in that the configurations of the first
- and of the second resonator $(70_1, 70_2)$ are identical,
- 4 with the resonators $(70_1, 70_2)$ being arranged axially
- 5 symmetrically with respect to one another with respect
- 6 to an axis of symmetry (73) which is at right angles to
- 7 the common oscillation axis (72).
- 1 3. The Coriolis gyro (1') as claimed in claim 1 or 2,
- 2 characterized in that the first oscillators $(3_1, 3_2)$ are
- 3 each connected by means of first spring elements
- 4 $(5_1 5_8)$ to a gyro frame $(7_1 7_{14})$ of the Coriolis gyro,
- 5 and the second oscillators $(4_1, 4_2)$ are each connected
- 6 by second spring elements $(6_1 6_4)$ to one of the first
- 7 oscillators $(3_1, 3_2)$.

- 4. The Coriolis gyro (1') as claimed in claim 3,
- 2 characterized in that the second oscillators $(4_1, 4_2)$
- 3 are attached/clamped in at one end to the first
- 4 oscillators $(3_1, 3_2)$ by means of the second spring
- 5 elements $(6_1 6_4)$ and/or the first oscillators $(3_1, 3_2)$
- 6 are attached/clamped in at one end to a gyro frame of
- 7 the Coriolis gyro by means of the first spring elements
- $(5_1 5_8)$.
- 1 5. The Coriolis gyro (1') as claimed in claim 3 or 4,
- 2 characterized by a device for production of
- 3 electrostatic fields, by means of which the alignment
- 4 angle of the first spring elements $(5_1 5_8)$ with
- 5 respect to the gyro frame can be varied, and/or the
- 6 alignment angle of the second spring elements $(6_1 6_4)$
- 7 with respect to the first oscillators $(3_1, 3_2)$ can be
- 8 varied.

- 1 6. The Coriolis gyro (1') as claimed in claim 5,
- characterized by
- 3 a device $(10_1 10_8, 11_1 11_4)$ by means of which it
- 4 is possible to determine first signals for the rotation
- 5 rate and quadrature bias, which occur within the first
- resonator (70_1) , and second signals for the rotation
- 7 rate and quadrature bias, which occur in the second
- 8 resonator (70_2) ,
- 9 control loops (60 67) by means of which the
- 10 alignment/strength of the electrostatic fields is
- 11 regulated such that the first and the second quadrature
- 12 bias are each as small as possible, and
- 13 a computation unit, which uses the first and
- second signals to determine the rotation rate, and uses
- an in-phase component of the electrostatic fields which
- 16 compensate for the first and second quadrature biases
- 17 to determine the acceleration to be measured.

7. A method for selective or simultaneous measurement

2 of rotation rates and accelerations using a rotation

3 rate Coriolis gyro (1') which has a first and a second

4 resonator $(70_1, 70_2)$ which are each in the form of a

5 coupled system comprising a first and a second linear

oscillator $(3_1, 3_2, 4_1, 4_2)$, with the rotation rates

7 being determined by tapping and evaluation of the

8 deflections of the second oscillators $(4_1, 4_2)$, having

9 the following steps:

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10 - the two resonators $(70_1, 70_2)$ are caused to carry

out oscillations in antiphase with one another along a

12 common oscillation axis (72),

13 - the deflections of the second oscillators $(4_1, 4_2)$

are compared with one another in order to determine an

antiphase deflection component which is a measure of

the rotation rate to be measured and/or in order to

determine a common in-phase deflection component, which

is a measure of the acceleration to be measured,

19 - calculation of the rotation rate/acceleration to

be measured from the in-phase deflection

21 component/antiphase deflection component.

- 1 8. The method as claimed in claim 7,
- 2 characterized in that the common in-phase deflection
- 3 component is determined as follows:
- 4 a first quadrature bias is determined which occurs
- 5 within the first resonator (70_1) ,
- 6 a second quadrature bias is determined which
- 7 occurs within the second resonator (70_2) ,
- 8 the first quadrature bias is calculated using the
- 9 second quadrature bias in order to determine a common
- quadrature bias component which is proportional to the
- 11 acceleration to be measured and represents the common
- in-phase deflection component.
- 1 9. The method as claimed in claim 8,
- 2 characterized in that electrostatic fields are produced
- 3 in order to vary the mutual alignment of the first and
- 4 second oscillators $(3_1, 3_2, 4_1, 4_2)$, with the
- 5 alignment/strength of the electrostatic fields being
- 6 regulated such that the first and the second quadrature
- 7 bias are each as small as possible.

Method for measurement of rotation rates/accelerations
using a rotation rate Coriolis gyro, as well as a
Coriolis gyro which is suitable for this purpose

ABSTRACT

A Coriolis gyro includes (1') has a first and a second resonator, $(70_1, 70_2)$, which are each in the form of a coupled system comprising a first and a second linear oscillators. $(3_1, 3_2, 4_1 \text{ and } 4_2)$, in which case The first resonator (70_1) together with (70_2) can be caused to oscillate in antiphase with respect to one another the second resonator along a common oscillation axis. (72) A system which is coupled in this way has the advantage that it is possible to measure the rotation rate and acceleration accelerations simultaneously, with insensitivity to disturbances and that it is insensitive to disturbance influences, for example (e.g., externally or internally acting vibrations).

(Figure 3)